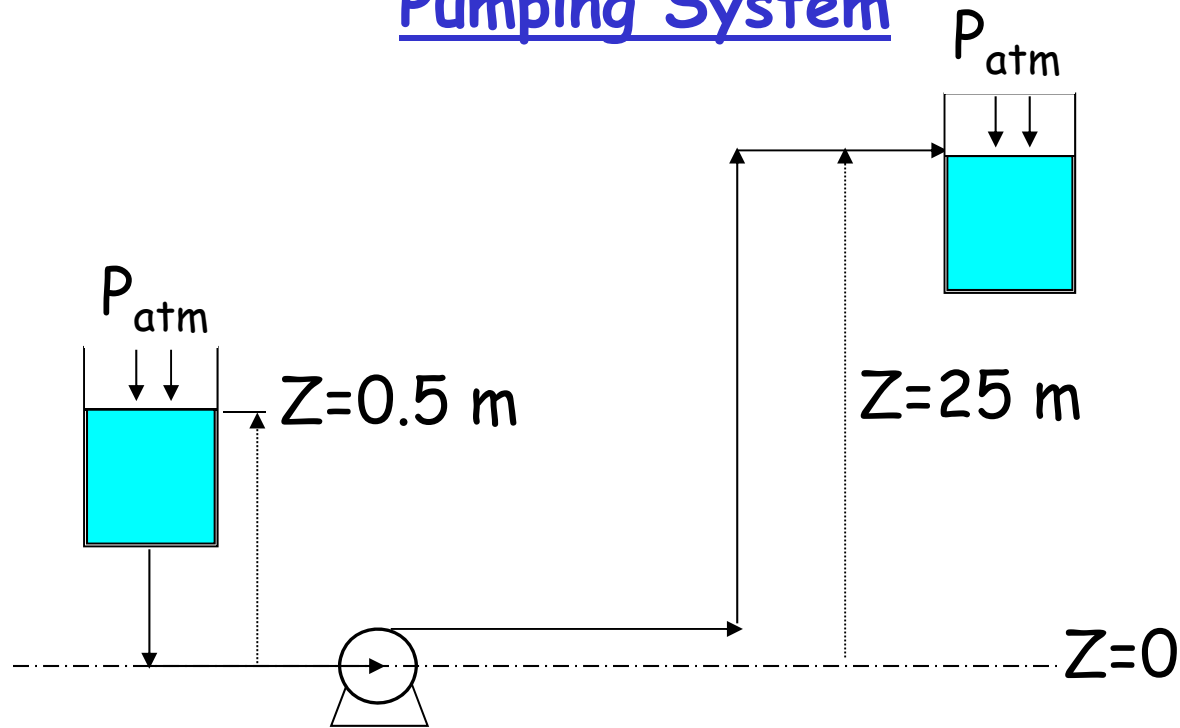


Pumping System



Suction line length (including fittings equivalent lengths) 12.0 m
Discharge line length (including fittings equivalent lengths) 50.0 m
ID of the suction pipe 0.4064 m, ID of the discharge line 0.3556 m
Water flow at $0.75 \text{ m}^3/\text{min}$.

The Fanning friction factor can be assumed at 0.008.



Pumping System

- (a) Find the friction loss in the suction side of the pump.
- (b) What is the pump inlet pressure?
- (c) What is the power necessary to drive the pump?

1 atm= 101,325 Pascal. $\rho=1,000 \text{ kg/m}^3$, $g=9.81 \text{ m/s}^2$.

Use the Bernoulli Equation:

$$g(Z_2 - Z_1) + \left(\frac{p_2}{\rho_2} - \frac{p_1}{\rho_1} \right) + \left(\frac{V_2^2 - V_1^2}{2} \right) = \Delta W_o - \sum F$$



Pumping System

(a) Find the friction loss in the suction side of the pump.

The friction loss is given by:

$$F = \frac{2V^2 f}{D} L$$

Need to calculate velocity:

$$V = \left(\frac{4}{\pi D^2} \right) Q = \left(\frac{4}{\pi 0.4064^2} \right) .75 \text{ m}^3/\text{min} = 0.09636 \text{ m/s}$$

$$\text{and } F = 2 * .09636^2 * 0.008 * 12 / 0.4064 = 0.0044 \text{ m}^2/\text{s}^2$$



Pumping System

(b) What is the pump inlet pressure?

Use the Bernoulli Equation for the suction line:

$$g(Z_2 - Z_1) + \left(\frac{p_2}{\rho_2} - \frac{p_1}{\rho_1} \right) + \left(\frac{V_2^2 - V_1^2}{2} \right) = \Delta W_o - \sum F$$

So, $g^*(-0.5) + (p_2 - p_1)/\rho + V_2^2/2 = -F$

and $p_2 = 1,000 (-0.0044 + 9.81*0.5 - .09636^2/2) \text{ kg / (m s}^2) +$

$101,325 \text{ Pa} = 106216 \text{ Pa or } 106.216 \text{ kPa absolute.}$

Think about cavitation!



Pumping System

(c) What is the power necessary to drive the pump?

Use the Bernoulli Equation between the end points:

$$g(Z_2 - Z_1) + \left(\frac{p_2}{\rho_2} - \frac{p_1}{\rho_1} \right) + \left(\frac{V_2^2 - V_1^2}{2} \right) = \Delta W_o - \sum F$$

So, $g*24.5 = \Delta W_o - (F_s + F_d)$

$$F_d = 2 * .125^2 * 50 * 0.008 / 0.3556 = 0.0356 \text{ m}^2/\text{s}^2$$

$$\text{and } \Delta W_o = 9.81 * 24.5 + (0.0044 + 0.0356) = 240.38 \text{ m}^2/\text{s}^2$$

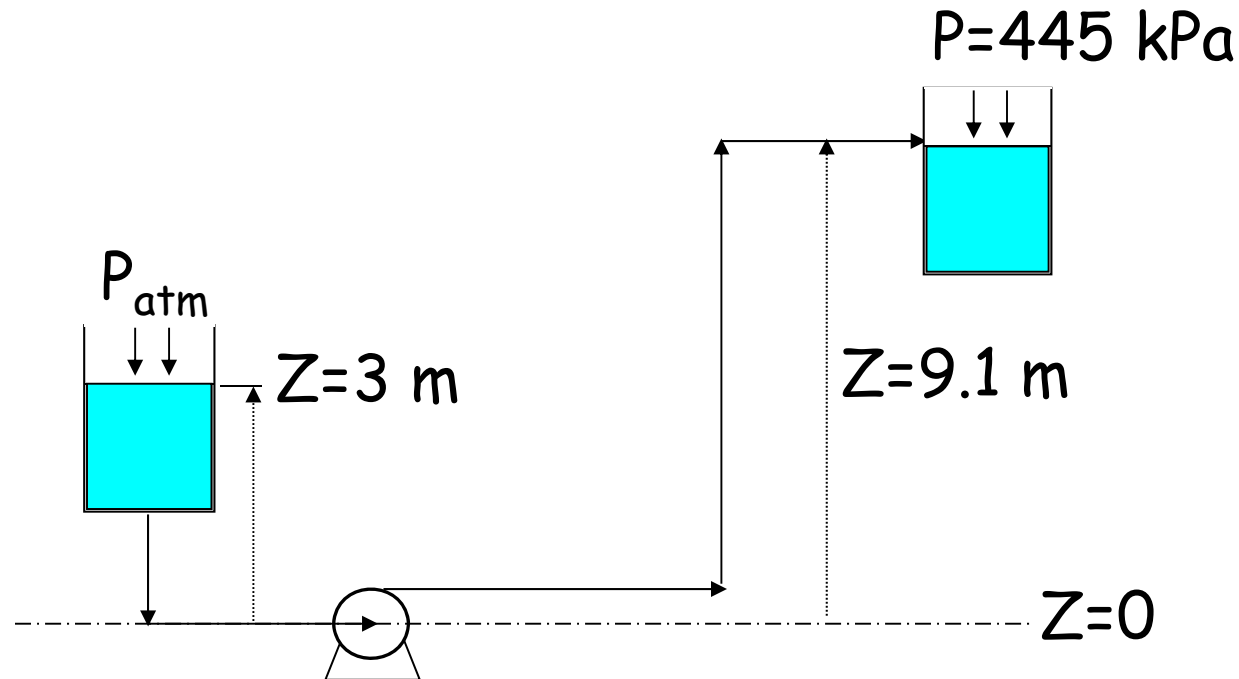
But Bernoulli above is per unit mass, so:

$$240.38 \text{ m}^2/\text{s}^2 * 1,000 \text{ kg}/\text{m}^3 * 0.75 \text{ m}^3 / 60 \text{ s} = 3.004 \text{ kW}$$

Actual power with 75% efficiency: $P = 3.004 / 0.75 = 4.00 \text{ kW}$



Pumping System (Problem 12-1 in PT&W)



Suction and discharge lines combined length of 46.0 m

Five 90° elbows in the lines,

ID of the pipe 0.078 m,

Lean oil flow at 2.7 kg/s.

Viscosity 15 cP, density 857 kg/m^3

What is the pump motor power, when the efficiency is 40% ?.



Pumping System

(a) Find the friction loss in the pipe system.

The friction loss is given by:

$$F = \frac{2V^2 f}{D} L$$

Need to calculate the Fanning factor and the equivalent lengths.

$$V = \frac{4}{\pi D^2} Q = \frac{4}{\pi (0.078)^2} * 2.7 / 857 \text{ m/s} = 0.659 \text{ m/s}$$

$$\text{So, Reynolds number} = \frac{D V \rho}{\mu} = 2,937$$

From Moody diagram (Fig 12-1) and steel : $f=0.011$

$$\text{Equivalent length (Table 12-1): } 5 * 32 * 0.078 + 46 = 58.5 \text{ m}$$

$$\text{and } F = 2 * 0.659^2 * 0.011 * 58.5 / 0.078 = 7.17 \text{ m}^2/\text{s}^2$$



Pumping System

(b) What is the power necessary to drive the pump?

Use the Bernoulli Equation between the end points:

$$g(Z_2 - Z_1) + \left(\frac{p_2}{\rho_2} - \frac{p_1}{\rho_1} \right) + \left(\frac{V_2^2 - V_1^2}{2} \right) = \Delta W_o - \sum F$$

So,

$$g \cdot 6.1 + (445,000 - 101,359)/857 + 0.659^2/2 = \Delta W_o - 7.17$$

$$\Delta W_o = 9.81 \cdot 6.1 + 7.17 + 400.98 + 0.22 = 468.21 \text{ m}^2/\text{s}^2$$

But Bernoulli is per unit mass, so:

$$468.21 \text{ m}^2/\text{s}^2 * 2.7 \text{ kg/s} = 1.265 \text{ kW}$$

Actual power with 40% efficiency: $P = 1.265/0.4 = 3.18 \text{ kW}$



Pressure loss

What is the pressure loss when 2.14 kg/s of benzene at 40° C flows in a 21 m long pipe with ID = 0.0409 m ?

The pipeline has six 90° elbows, one tee used as an elbow (equiv. resistance is 60 pipe diameters), one globe valve, and one gate valve.

Density 849 kg/m³, viscosity at 40° C is 5x10⁻⁴ Pa·s



Pressure loss

Need to calculate the Fanning factor and the equivalent lengths.

Equivalent lengths (Table 12-1):

6	90° elbows	192 D
1	Tee	60 D
1	Gate valve (open)	7 D
1	Globe valve (open)	300 D

So, $L_{eq} = 559 * 0.0409 = 22.9$ m, and $L = 22.9 + 21 = 43.9$
 $V = (4/\pi D^2) Q = (4/\pi 0.0409^2) * 2.14 / 849$ m/s = 1.92 m/s

$$Re = D V \rho / \mu = 1.33 \times 10^5$$

From Moody diagram (Fig 12-1) and steel : $f = 0.0053$

and $F = 2 * 1.92^2 * 0.0053 * 43.9 / 0.0409 = 41.9$ N m/kg

Finally, $\Delta P = F \rho = 41.9 * 849 = 35,575$ Pa

